

SQUEEZE FILM-BEARING THROUGH DIFFERENT POROUS CONSTRUCTIONS: ASSOCIATION OF CHANGED MODELS

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ABSTRACT

Squeeze film geometry of truncated cone. Lower porous plate squeeze film orientation of various shapes (annular, round, elliptic, rectangular and cone) utilizing Morgan-Cameron guess. The impacts of the state of plate and porosity on the bearing execution are determined. The ferrofluid based squeeze film for round and conical orientation. The attractive field considered was the transverse way of the fluid flow. Here, they have considered Shliomis model to tackle the issue since it dealt with pivot of the fluid particles just as fluid. The subsequent overseeing conditions are nonlinear coupled conditions and is comprehended utilizing annoyance strategy regarding dimensionless Brownian unwinding time parameter. The impact of attractive fluid parameters on different bearing attributes is examined numerically. This article examined porous truncated cone squeeze film-bearing model thinking about the impacts of porosity, penetrability, squeeze speed and slanted variable attractive field. Impacts of two porousness models-globular circle and narrow gaps are additionally talked about. Articulations for pressure and burden conveying capacity are acquired. The outcomes for dimensionless burden conveying capacity are figured.

Keyword: Porus, Flim, ferrofluid, fluid.

INTRODUCTION

The different structured heading like round, annular, elliptic, conical, and so on. It is demonstrated that with the increment of polarization parameter u^* , the heap conveying capacity increments. In this way, closed the prevalence execution of the course with MF as lubricant. It is additionally presumed that the holding on for MF can bolster a heap notwithstanding when there is no flow. The squeeze film dependent on attractive fluid for conical plates. There they found that the presentation of the holding on for this lubricant is moderately superior to the traditional lubricant. Likewise, it is discovered that the negative impact incited purchase the porosity can be kill by the beneficial outcome brought about by the polarization parameter. Further, the paper recommends about the picking of appropriate blend of the polarization parameter and semi-vertical plot for upgrading bearing exhibitions. The down execution of hydromagnetic squeeze film between two directing truncated conical plates. The plates are viewed as electrically leading and the leeway space between them is filled by an electrically directing lubricant. A uniform transverse attractive field is applied between the plates. The subsequent Reynolds condition is comprehended for pressure, load conveying

capacity and reaction time. The outcomes propose better exhibitions for the bearing when contrasted with ordinary lubricant.

The longitudinal roughness impact on MF based squeeze film between conical plates. It is demonstrated that the exhibition of the bearing gets improved because of negative slanted roughness. Likewise, it is demonstrated that the standard deviation builds the heap conveying capacity which is not normal for the instance of transverse surface roughness. The impacts of fluid inactivity powers on the squeeze film qualities of conical plates utilizing ferrofluid lubricant. By applying the found the middle value of force standard, a grease condition administering the film pressure is determined. Contrasting and the non-idleness non-attractive case, better squeeze exhibitions are anticipated when working with enormous estimation of the inertial parameter of fluid dormancy powers.

CALCULATION OF LOAD CARRYING CAPACITY

Load carrying capacity

$$W = 2\pi \int_{b \operatorname{cosec} \omega}^{a \operatorname{cosec} \omega} px \, dx \quad (1)$$

Signifies

$$W = \frac{\pi\mu_0\bar{\mu}K(a^2 - b^2)^2 \operatorname{cosec}^4 \omega \left[\frac{(a/b)^2 + 1}{(a/b)^2 - 1} - \frac{2(a/b)}{(a/b)^2 - 1} \right]}{12} - \frac{3\pi\eta\dot{h} \operatorname{cosec}^3 \omega (a^2 - b^2)^2 \left[\frac{(a/b)^2 + 1}{(a/b)^2 - 1} - \frac{1}{\ln(a/b)} \right]}{2 \left(h^3 \sin^3 \omega + \frac{\varepsilon^3 D_c^2}{15(1-\varepsilon)^2} H^* \right)}$$

(2)

It is written in dimensionless structure as

$$\bar{W} = -\frac{Wh^3}{\eta\dot{h}\pi^2(a^2 - b^2)^2 \operatorname{cosec}^4 \omega} = \frac{\mu^* \left[\frac{(a/b)^2 + 1}{(a/b)^2 - 1} - \frac{2(a/b)}{(a/b)^2 - 1} \right]}{12\pi} + \frac{3 \operatorname{cosec}^2 \omega \left[\frac{(a/b)^2 + 1}{(a/b)^2 - 1} - \frac{1}{\ln(a/b)} \right]}{2\pi \left(1 + \frac{\varepsilon^3}{15(1-\varepsilon)^2} \psi \operatorname{cosec}^3 \omega \right)} \quad (3)$$

MATHEMATICAL FORMULATION OF GLOBULAR SPHERE - PERMEABILITY MODEL

Figure 1 shows schematic chart of porous truncated cone squeeze film-bearing. The lower surface is connected with a porous matrix of thickness H^* . By the standard suspicions of grease hypothesis, disregarding latency terms, subsidiaries of fluid speed over the film prevail and consolidating conditions the condition administering the pressure dissemination p in the film locale utilizing ferrofluid (FF) as lubricant fulfills the altered condition.

$$\frac{\partial}{\partial x} \left[h^3 \frac{\partial}{\partial x} \left(p - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) \right] + \frac{\partial}{\partial y} \left[h^3 \frac{\partial}{\partial y} \left(p - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) \right] = 12\eta\dot{h} - 12\eta w|_{z=0} \quad (4)$$

With

$$\frac{\partial^2}{\partial x^2} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) + \frac{\partial^2}{\partial y^2} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) + \frac{\partial^2}{\partial z^2} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) = 0. \quad (5)$$

where x, y, z are the Cartesian coordinates, h is the film thickness, H is quality of variable attractive field, is the squeeze velocity of the upper bearing surface, w0 is the h = dh/dt z-segment of the fluid speed at z = 0, is fluid viscosity, 0 is the free space porousness, is attractive powerlessness and P is pressure in the porous locale.

Coordinating condition as for z over the porous matrix thickness (-H *, 0) yields

$$\left[\frac{\partial}{\partial z} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) \right]_{z=-H^*}^{z=0} = - \int_{-H^*}^0 \left[\frac{\partial^2}{\partial x^2} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) + \frac{\partial^2}{\partial y^2} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) \right] dz. \quad (6)$$

Since

$$\left[\frac{\partial}{\partial z} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) \right]_{z=-H^*} = 0, \quad (7)$$

as z = -H * is a strong surface, that is the porous matrix is press-fitted with a strong lodging as appeared in Figure 1.

Utilizing conditions (6) and (7)

$$\left[\frac{\partial}{\partial z} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) \right]_{z=0} \approx -H^* \left[\frac{\partial^2}{\partial x^2} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) + \frac{\partial^2}{\partial y^2} \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) \right], \quad (8)$$

MATHEMATICAL FORMULATION OF CAPILLARY FISSURES - PERMEABILITY MODEL

At the point when the porous matrix is planned with slender fissures made out of three arrangements of commonly orthogonal fissures as recommended by Irmay, at that point condition (8) moves toward becoming

$$\nabla^2 \left(P - \frac{1}{2} \mu_0 \bar{\mu} H^2 \right) = \frac{12\eta m \dot{h}}{mh^3 + (1 - m^{\frac{2}{3}})(1 - m^{\frac{1}{3}})^2 D_s^2 H^*}, \quad (9)$$

$$m = 1 - \varepsilon = \frac{a^3}{(a + b)^3}$$

Where Ds is a mean strong size and

The heap conveying capacity in dimensionless structure can be gotten as

$$\bar{W} = - \frac{Wh^3}{\eta \dot{h} \pi^2 (a^2 - b^2)^2 \operatorname{cosec}^4 \omega} = \frac{\mu^*}{12\pi} \left[\frac{(a/b)^2 + 1}{(a/b)^2 - 1} - \frac{2(a/b)}{(a/b)^2 - 1} \right] + \frac{3 \operatorname{cosec}^2 \omega}{2\pi \left(1 + \frac{(1 - m^{\frac{2}{3}})(1 - m^{\frac{1}{3}})^2}{m} \psi \operatorname{cosec}^3 \omega \right)} \left[\frac{(a/b)^2 + 1}{(a/b)^2 - 1} - \frac{1}{\ln(a/b)} \right], \quad (10)$$

Where

$$\mu^* = - \frac{\mu_0 \bar{\mu} K h^3}{\eta \dot{h}}, \quad \psi = \psi_f = \frac{D_s^2 H^*}{h^3}.$$

Table 1 demonstrates the consequences of W for porous truncated cone squeeze film-bearing

considering $\bar{W} = -Wh^3 / \eta h \pi^2 (a^2 - b^2)^2 \text{cosec}^4 \omega$

It is seen from the table that porous truncated cone doesn't bolster load when dimensionless magnetization parameter μ^* and dimensionless penetrability parameter ψ 0.0001, that is when there is no utilization of FF as lubricant. Be that as it may, when FF is utilized as lubricant, at that point it supports burden and this impact is progressively obvious as μ^* increments. In addition, the turnaround pattern of \bar{W} is seen with deference to ψ , that is when contrasted with different past bearing structures here \bar{W} increments with the expansion of ψ .

RESULTS & DISCUSSION

5.1 Comparison between Globular Sphere And Capillary Fissures Models Of Truncated Cone With Other Bearing Designs

At the point when the porous matrix is structured with globular circles proposed. At that point the penetrability of the porous matrix joined at the lower plate or plate is characterized by

$$\frac{\epsilon^3 D_c^2}{180 (1 - \epsilon)^2},$$

Where D_c is a mean molecule size, is the porosity of the porous matrix. At the point when the porous matrix structured with slim fissures made out of three arrangements of commonly orthogonal fissures as recommended. At that point the porousness of the porous matrix connected at the lower plate is characterized by

$$\frac{(1 - m^{\frac{2}{3}})(1 - m^{\frac{1}{3}})^2 D_s^2}{12m},$$

$$m = 1 - \epsilon = \frac{a^3}{(a + b)^3}.$$

Where D_s is a mean strong size and ϵ . The effect of these two porous structures on W is appeared in Table 5.2 for the accompanying estimation of various parameters.

$$\epsilon = 0.2, D_c = D_s = 0.00001(m), H^* = 0.0001(m),$$

$$h = 0.000005(m), \omega = \pi / 6 \text{ (rad.)}.$$

From the Table 2, it is seen that better burden conveying capacity can be acquired when the porous matrix is planned with globular porous structures aside from truncated cone bearing structure framework. For truncated cone bearing plan framework narrow fissures structures have better impact on \bar{W} .

Reynolds-type condition for squeeze film-bearing truncated cone structure framework is hypothetically gotten by considering conditions from ferrohydrodynamic hypothesis by R.E. Rosensweig and condition of congruity in film just as porous region.

The legitimacy of the Darcy's law is accepted in the porous region. The impacts of porosity, porousness, squeeze velocity and variable attractive field are considered. The variable

attractive field considered here is slanted to the lower circle or plate. In addition, the porous surface is considered in light of its favorable property of self-grease and no need of outside lubricant supply.

It is reasoned that the porous truncated cone bearing doesn't bolster load on account of regular lubricant, though it does on account of FF as lubricant for littler estimations of additionally, the invert pattern of $\frac{-}{w}$. Can be seen with deference to as contrasted with other bearing plan frameworks. Also, the better execution (in the feeling of W increments) of the direction are seen on account of globular circle model recommended by Kozeny-Carman for penetrability in the porous region aside from on account of truncated cone bearing structure, where better burden conveying capacity can be gotten on account of fine fissures model.

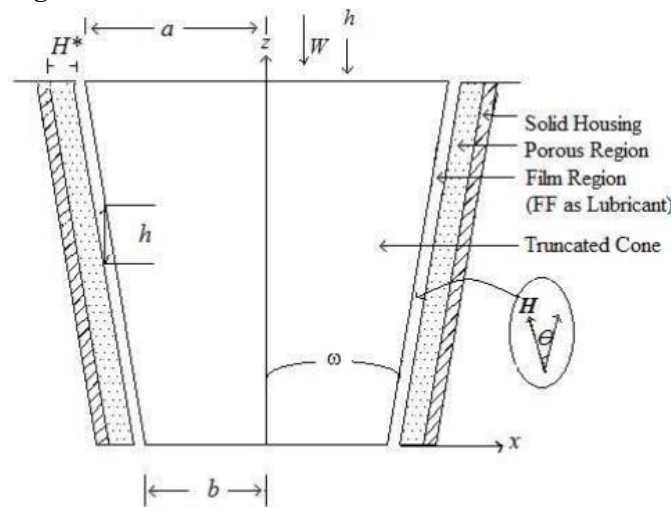


Fig 1 Schematic diagram of porous truncated cone squeeze film geometry

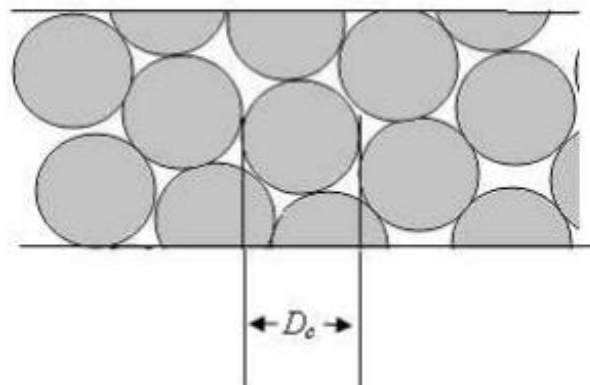


Fig 2 Globular sphere model of porous matrix suggested by Kozeny-Carman

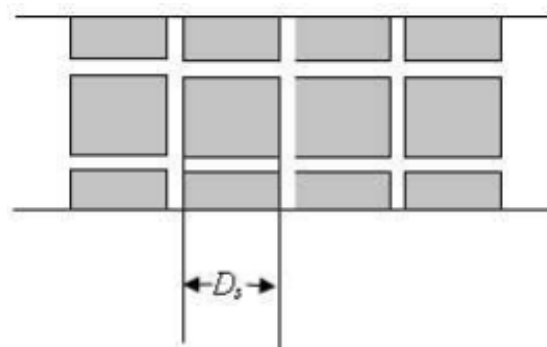


Fig 3 Capillary fissures model of porous matrix suggested by Irmay

Table 1: Comparison of dimensionless load carrying capacity \bar{w} for $\omega = \dot{h}/6$

Dimensionless permeability parameter	W		
	$\mu^* = 0$	$\mu^* = 0.2$	$\mu^* = 0.3$
$\psi = 0.0001$	0.00000	0.01745	0.02618
$\psi = 0.01$	0.00003	0.01749	0.02622
$\psi = 1.0$	0.19970	0.21716	0.22588

Table 2: Comparison between globular sphere model and capillary fissures model

Bearing geometry	W					
	Annular	Circular	Infinitely Rectangular ($a \square b$)	long	Complete cone	Truncated cone
Globular sphere model	9.1364	4.4628	1.9241		12.2885	0.0262
Capillary fissures model	0.8765	0.3650	0.1849		0.1984	0.2469

At the point when two lubricated surfaces approach each other with a typical speed (known as squeeze speed), at that point squeeze film wonder emerge .Investigation of squeeze film conduct are seen in numerous fields of genuine, for example, in machine instruments, gears, moving components, water driven frameworks, motors, grip plates, and so forth. Additionally, it is seen in the investigation of human knee joints and other skeletal joints as bio-oil .Squeeze film with the connection of porous layer (area or plate or network or surface) are generally utilized in industry on account of its profitable property of self-oil and no need of outside grease supply.

As of late, numerous hypothetical and exploratory creations are made on the bearing structure frameworks just as on the greasing up substances so as to expand the productivity of the bearing exhibitions. One of the significant upsets toward greasing up substances is an innovation of ferrofluids (FFs). Numerous scientists have additionally attempted to discover its application as ointment in squeeze film bearing structure frameworks. The impacts of attractive liquid (MF) on squeeze film bearing structure framework under a remotely applied attractive field sideways to the lower surface with the lower porous surface made out of three slender layers with various porosities. Express answers for speed, pressure, load conveying limit and reaction time are gotten. It is discovered that upper plate sets aside longer effort to descend for this situation when contrasted with customary ointment based squeeze film. Kumar et. Squeeze film for round and funnel shaped course utilizing ferrofluid (FF) as ointment with the impacts of turn of particles and consistent attractive field transverse way and numerically contemplated different bearing qualities. Bended porous circular circles squeeze film with the impact of MF and demonstrated that pressure, load conveying limit and reaction time increments with the expansion of charge.

Films with MF impact and demonstrated the predominance execution of the MF ointment than regular oil. Squeeze film between porous annular bended plates with the impacts of rotational dormancy just as MF and found that the expansion in pressure and load conveying limit depended distinctly on the charge while reaction time reliant on polarization, liquid idleness and speed of pivot of the plates. Squeeze film in a long diary bearing utilizing FF as grease and found that load conveying limit and reaction time expanded with the expanding estimations of the whimsy proportion. MF based squeeze film between porous funnel shaped plates and found

that negative impact instigated by the porosity can be killed by the beneficial outcome brought about by the polarization parameter. MF based squeeze film for truncated cone shaped plates with the impact of longitudinal unpleasantness and found that load conveying limit can be expanded with charge just as adversely slanted harshness. The pressure and reaction time likewise found to increment with charge. Different porous structures on bended porous circular plates squeeze film utilizing FF as oil and found that globular sphere model have more effect on increment of load conveying limit as contrast with narrow gaps model.

Read squeeze film attributes for tapered plates with the impact of liquid idleness and FF, and demonstrated the better execution of the framework when contrasted with non-dormancy non-attractive case. Squeeze film qualities of parallel circular plates with the impacts of FF and non-Newtonian couple-stresses utilizing transverse attractive field. With these impacts, it was demonstrated that higher load conveying limit and extends moving toward time acquired.

CONCLUSION

The motivation behind the present paper is to consider recently structured squeeze film bearing made up of a sphere and a level plate utilizing water based FF as oil, which is constrained by sideways and variable attractive field, with the impacts of porosity, slip speed and squeeze speed. With these impacts, the effect of squeeze film tallness, penetrability and width of the porous layer are contemplated. Articulations for pressure, load conveying limit and reaction time are acquired from Reynolds condition. The dimensionless pressure conveyance, load conveying limit and reaction time are determined and exhibited graphically.

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